



POSTAL BOOK PACKAGE 2025

MECHANICAL ENGINEERING

CONVENTIONAL Practice Sets

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FLUID MECHANICS AND FLUID MACHINERY

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Reciprocating Pump

Q1 A double acting reciprocating pump having piston area 0.16 m^2 has a stroke 0.40 m long. The discharge of water through the pump is 6.0 m^3 per minute at a speed of 50 rpm through a height of 12 m . Find the slip of the pump and power required to drive the pump.

Solution:

Given : $N = 50 \text{ rpm}$; $Q_{\text{act}} = \frac{6}{60} = 0.1 \text{ m}^3/\text{s}$; $L = 0.40 \text{ m}$

$$h = h_s + h_d = 12 \text{ m}$$

$$A = 0.16 \text{ m}^2$$

Theoretical discharge,

$$Q_{\text{th}} = \frac{2ALN}{60} = \frac{2 \times 0.16 \times 0.40 \times 50}{60} = 0.11 \text{ m}^3/\text{s}$$

\therefore

$$\text{Slip} = Q_{\text{th}} - Q_{\text{ac}}$$

$$= 0.11 - 0.10 = 0.01 \text{ m}^3/\text{s}$$

Power Required,

$$P = \frac{2 \times S_g \times ALN \times (h_s + h_d)}{60000} \text{ kW}$$

$$= \frac{2 \times 10^3 \times 9.81 \times 0.16 \times 0.4 \times 50 \times 12}{60000} \text{ kW} = 12.56 \text{ kW}$$

Q2 A single acting reciprocating pump has a plunger diameter 200 mm and stroke length 300 mm . It draws water from a sump 3.6 m below the pump centre line with a pipe 4.8 m long and 180 mm in diameter and lift water to a location 20 m above the pump with a pipe 25 m long and 100 mm diameter. The pump is driven with SHM. If the atmospheric pressure head is 10.3 m of water and separation occurs at 2.6 m of water absolute. Find the maximum operational speed of the pump.

Solution:

Given:

For reciprocating pump,

Length of suction pipe, $l_s = 4.8 \text{ m}$

Diameter of suction pipe, $d_s = 180 \text{ mm} = 0.18 \text{ m}$

Area of suction pipe, $a_s = \frac{\pi}{4} d_s^2 = \frac{\pi}{4} \times (0.18)^2 = 0.0254 \text{ m}^2$

Diameter of plunger, $D = 200 \text{ mm} = 0.2 \text{ m}$

Area of plunger, $A = \frac{\pi}{4} D^2 = \frac{\pi}{4} \times (0.2)^2 = 0.03141 \text{ m}^2$

Length of delivery pipe, $l_d = 25 \text{ m}$

Diameter of delivery pipe, $d_d = 100 \text{ mm} = 0.1 \text{ m}$

Area of delivery pipe, $a_d = \frac{\pi}{4} \times (0.1)^2 = 7.85 \times 10^{-3} \text{ m}^2$

Head for separation, $H_{\text{sep}} = 10.3 - 2.6 = 7.7 \text{ m}$

$$H_{as} = \frac{L_s}{g} \times \frac{A}{a_s} \times \omega^2 r \cos \theta$$

where

A = area of cross-section of piston,

a_s = area of cross-section of suction pipe

ω = angular velocity = $\frac{2\pi N}{60}$

r = radius of crank = $\frac{L}{2} = \frac{180}{2} = 90$ mm

θ = angular distance travelled by crank.

$$\therefore H_{as} = \frac{L_s}{g} \times \frac{\pi D^2}{4} \times \frac{4}{\pi (d_s)^2} \times \omega^2 r \cos \theta$$

$$\Rightarrow H_{as} = \frac{L_s}{g} \times \left(\frac{D}{d_s}\right)^2 \times \omega^2 r \cos \theta$$

The head loss due to friction in suction pipe is given by

$$H_{fs} = \frac{f_s L_s}{2g d_s} \times \left(\frac{A}{a_s} \omega r \sin \theta\right)^2$$

$$\Rightarrow H_{fs} = \frac{f_s L_s}{2g d_s} \times \left(\frac{D}{d_s}\right)^4 (\omega r \sin \theta)^2$$

Pressure head at the beginning of stroke

$$H_{ts} = H_{atm} - H_{as} - H_{fs} - H_s$$

At the beginning,

$$\theta = 0^\circ$$

$$\therefore H_{as} = \frac{6}{9.81} \times \left(\frac{120}{90}\right)^2 \times \left(\frac{2\pi \times 72}{60}\right)^2 \times 90 \times 10^{-3} \times \cos 0^\circ$$

$$\Rightarrow H_{as} = 5.56 \text{ m}$$

Also,

$$H_{fs} = \frac{0.022 \times 6}{2 \times 9.81 \times 90 \times 10^{-3}} \times \left(\frac{120}{90}\right)^4 \times \left(\frac{2\pi \times 72}{60} \times 90 \times 10^{-3} \times \sin 0^\circ\right)^2$$

$$\Rightarrow H_{fs} = 0$$

In terms of water, $H_{atm} = 10.3$ m of water

$$\therefore H_{ts} = 10.3 - 5.56 - 0 - 2.5 = 2.24 \text{ m}$$

Pressure head at the middle of stroke,

$$\theta = 90^\circ$$

$$\Rightarrow \cos 90^\circ = 0$$

and $\sin 90^\circ = 1$

$$\therefore H_{as} = 0$$

$$H_{fs} = \frac{0.022 \times 6}{2 \times 9.81 \times 90 \times 10^{-3}} \times \left(\frac{120}{90}\right)^4 \times \left[\frac{2\pi \times 72}{60} \times 90 \times 10^{-3}\right]^2 = 0.1088$$

$$\therefore H_{ts} = 10.3 - 0 - 0.1088 - 2.5 = 7.6912 \text{ m}$$

Pressure head at the end of stroke

$$\theta = 180^\circ$$

$$\Rightarrow \sin 180^\circ = 0$$

and $\cos 180^\circ = -1$

$$\therefore H_{as} = 5.56 \times \cos 180^\circ = -5.56 \text{ m}$$

$$H_{fs} = 0$$

$$\therefore H_{ts} = 10.3 - (-5.56) - 0 - 2.5 = 13.36 \text{ m}$$

